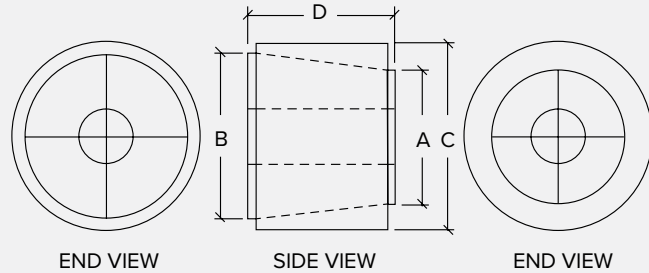


15-AC-D

Axial Cone Dissipative Silencer



Insertion Loss (IL)

+ : “forward flow” where noise & airflow move in same direction (e.g. supply side)

- : “reverse flow” where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
15	0	15	- 2000	1	2	8	13	11	7	5	4
			0	1	2	8	12	10	7	5	5
			+ 2000	1	2	8	11	10	7	5	4
15	0	30	- 2000	1	5	10	20	17	11	9	8
			0	1	5	10	20	17	10	9	8
			+ 2000	1	5	9	20	17	11	9	8
15	6	15	- 2000	1	2	8	13	15	13	10	8
			0	1	2	8	12	14	13	10	8
			+ 2000	1	2	8	12	14	13	10	9
15	6	30	- 2000	1	7	11	21	28	22	15	12
			0	1	6	10	20	27	21	15	12
			+ 2000	1	5	10	20	26	21	15	13
15	10	15	- 2000	1	3	10	16	21	21	15	13
			0	1	3	9	15	20	20	15	13
			+ 2000	1	3	9	14	19	20	15	13
15	10	30	- 2000	1	8	14	25	41	38	25	17
			0	1	7	13	24	39	37	25	18
			+ 2000	1	6	12	23	38	36	26	18

Pressure Drop (PD)

*Note: For pressure drop at other velocities Actual Pressure Drop =

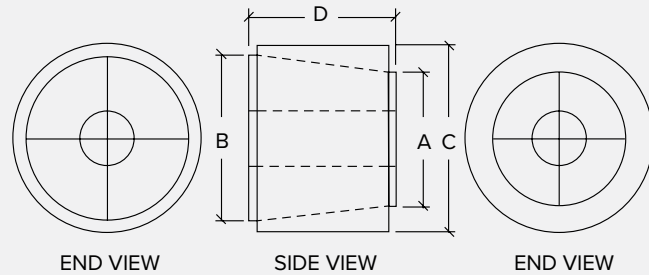
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}} \right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
15	20	25	0	15	65	0.03	0.04	0.03	0.13
				30	100	0.04	0.06	0.02	0.12
15	20	25	6	15	90	0.03	0.04	0.06	0.19
				30	130	0.03	0.05	0.05	0.18
15	20	25	10	15	100	0.03	0.04	0.12	0.30
				30	140	0.04	0.07	0.09	0.27

19-AC-D

Axial Cone Dissipative Silencer



Insertion Loss (IL)

+ : “forward flow” where noise & airflow move in same direction (e.g. supply side)

- : “reverse flow” where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
19	0	19	- 2000	1	2	8	13	10	7	5	5
			0	1	2	8	12	9	7	5	5
			+ 2000	1	2	7	11	9	7	5	5
19	0	38	- 2000	1	5	10	20	18	11	9	7
			0	1	5	10	20	17	10	9	7
			+ 2000	1	5	9	20	17	10	9	7
19	6	19	- 2000	1	2	8	14	15	14	10	8
			0	1	2	8	13	14	13	10	8
			+ 2000	1	2	7	13	13	13	10	8
19	6	38	- 2000	1	8	11	22	28	21	14	10
			0	1	7	10	21	26	20	14	10
			+ 2000	1	7	9	21	26	20	14	10
19	10	19	- 2000	1	3	10	16	19	19	15	13
			0	1	3	9	15	18	18	15	13
			+ 2000	1	3	8	14	17	18	15	13
19	10	38	- 2000	1	9	12	24	37	36	19	15
			0	1	9	11	23	36	35	19	15
			+ 2000	1	9	10	22	36	34	19	15

Pressure Drop (PD)

*Note: For pressure drop at other velocities Actual Pressure Drop =

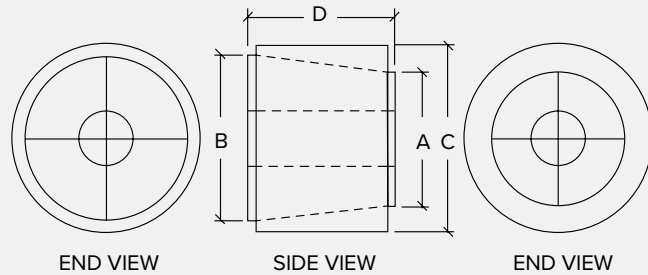
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}} \right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
19	24	30	0	19	95	0.03	0.05	0.10	0.15
				38	155	0.03	0.05	0.05	0.15
19	24	30	6	19	125	0.04	0.06	0.09	0.21
				38	190	0.05	0.08	0.07	0.19
19	24	30	10	19	140	0.05	0.08	0.10	0.25
				38	205	0.06	0.09	0.06	0.21

21-AC-D

Axial Cone
Dissipative Silencer



Insertion Loss (IL)

+ : **“forward flow”** where noise & airflow move in same direction (e.g. supply side)

- : **“reverse flow”** where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
21	0	21	- 2000	1	3	8	13	9	7	5	5
			0	1	3	8	12	9	7	5	5
			+ 2000	1	3	7	12	8	7	5	5
21	0	42	- 2000	1	6	10	21	16	11	9	8
			0	1	6	10	20	16	10	9	8
			+ 2000	1	6	9	20	16	11	9	8
21	6	21	- 2000	1	3	10	15	14	11	8	7
			0	1	3	9	14	13	11	8	7
			+ 2000	1	3	8	14	13	11	8	7
21	6	42	- 2000	1	8	12	21	23	17	14	13
			0	1	8	11	20	22	17	14	13
			+ 2000	1	8	11	20	22	17	14	13
21	8	21	- 2000	1	4	10	16	15	15	10	8
			0	1	4	10	15	14	15	10	8
			+ 2000	1	4	9	15	14	15	10	8
21	8	42	- 2000	1	9	13	22	28	22	15	14
			0	1	9	12	21	27	21	15	14
			+ 2000	1	9	12	21	27	21	15	14
21	12	21	- 2000	1	4	12	18	21	19	12	10
			0	1	4	10	17	20	19	12	10
			+ 2000	1	4	10	17	20	19	12	10
21	12	42	- 2000	1	10	14	24	37	34	18	16
			0	1	10	13	23	36	33	18	16
			+ 2000	1	10	13	23	35	33	18	16

Pressure Drop (PD)

*Note: For pressure drop at other velocities
Actual Pressure Drop =

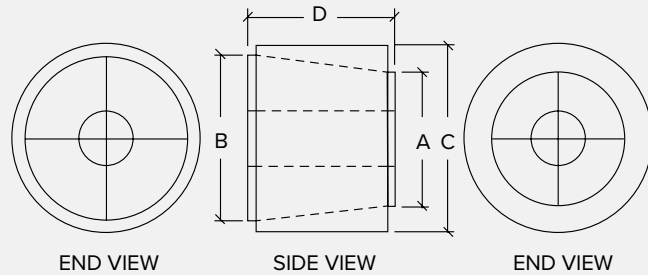
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}}\right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
21	26	32	0	21	115	0.03	0.05	0.11	0.21
			0	42	185	0.04	0.06	0.09	0.19
21	26	32	6	21	150	0.04	0.06	0.10	0.21
			6	42	225	0.04	0.07	0.08	0.19
21	26	32	8	21	155	0.04	0.06	0.10	0.22
			8	42	230	0.04	0.07	0.08	0.20
21	26	32	12	21	170	0.05	0.08	0.12	0.27
			12	42	250	0.06	0.10	0.11	0.26

25-AC-D

Axial Cone
Dissipative Silencer



Insertion Loss (IL)

+ : **“forward flow”** where noise & airflow move in same direction (e.g. supply side)

- : **“reverse flow”** where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
25	0	25	- 2000	1	4	8	13	9	7	6	4
			0	1	4	8	12	8	7	6	4
			+ 2000	1	4	7	12	9	7	6	4
25	0	50	- 2000	1	7	11	21	18	11	9	9
			0	1	7	10	20	17	10	9	8
			+ 2000	1	7	9	20	17	11	10	8
25	8	25	- 2000	1	5	9	14	15	11	9	7
			0	1	4	8	13	14	11	9	7
			+ 2000	1	4	7	12	13	11	9	7
25	8	50	- 2000	1	8	11	22	21	16	13	11
			0	1	8	10	21	20	15	13	11
			+ 2000	1	8	10	21	20	15	13	11
25	12	25	- 2000	1	5	10	17	19	14	11	9
			0	1	4	9	16	18	14	11	9
			+ 2000	1	4	9	15	17	14	11	9
25	12	50	- 2000	1	10	14	24	26	19	16	12
			0	1	9	13	23	25	18	16	12
			+ 2000	1	9	13	23	25	18	17	13
25	16	25	- 2000	1	6	12	20	22	20	14	12
			0	1	5	11	19	22	20	14	12
			+ 2000	1	5	10	18	21	20	15	11
25	16	50	- 2000	1	11	15	27	33	25	19	16
			0	1	10	14	26	32	25	19	16
			+ 2000	1	10	14	26	32	25	20	17

Pressure Drop (PD)

*Note: For pressure drop at other velocities
Actual Pressure Drop =

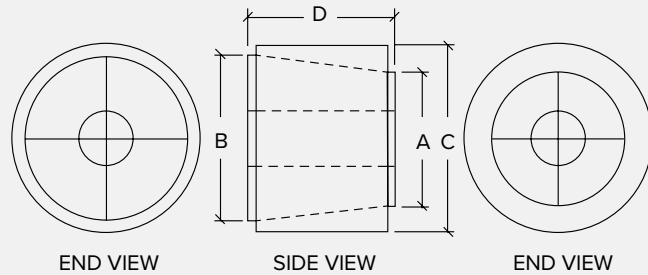
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}}\right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
25	32	37	0	25	160	0.04	0.06	0.11	0.21
			0	50	250	0.04	0.07	0.10	0.20
25	32	37	8	25	205	0.04	0.07	0.10	0.22
			8	50	305	0.05	0.08	0.09	0.21
25	32	37	12	25	220	0.04	0.06	0.08	0.22
			12	50	330	0.04	0.07	0.07	0.21
25	32	37	16	25	240	0.06	0.10	0.25	0.43
			16	50	255	0.07	0.11	0.23	0.41

29-AC-D

Axial Cone Dissipative Silencer



Insertion Loss (IL)

+ : **“forward flow”** where noise & airflow move in same direction (e.g. supply side)

- : **“reverse flow”** where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
29	0	29	- 2000	1	4	9	11	9	7	6	4
			0	1	4	9	10	8	7	6	4
			+ 2000	1	4	9	10	8	7	6	4
29	0	58	- 2000	1	7	15	20	15	10	10	8
			0	1	7	14	19	15	10	10	8
			+ 2000	1	7	14	20	15	11	11	8
29	12	29	- 2000	1	5	10	16	13	11	9	8
			0	1	5	9	15	13	11	9	8
			+ 2000	1	5	9	14	13	11	9	8
29	12	58	- 2000	1	9	16	25	24	18	15	13
			0	1	9	15	24	24	17	15	12
			+ 2000	1	9	15	24	25	17	16	13
29	16	29	- 2000	1	6	12	19	21	14	13	10
			0	1	5	11	18	20	14	12	10
			+ 2000	1	5	11	17	20	14	12	10
29	16	58	- 2000	1	10	17	29	30	24	18	16
			0	1	10	16	28	30	23	18	15
			+ 2000	1	10	16	27	30	24	19	16
29	20	29	- 2000	1	8	13	23	29	20	16	13
			0	1	7	12	22	28	20	15	12
			+ 2000	1	7	12	21	28	20	15	12
29	20	58	- 2000	2	12	19	33	35	30	21	19
			0	1	11	18	32	35	30	21	18
			+ 2000	2	11	18	31	35	31	22	19

Pressure Drop (PD)

*Note: For pressure drop at other velocities
Actual Pressure Drop =

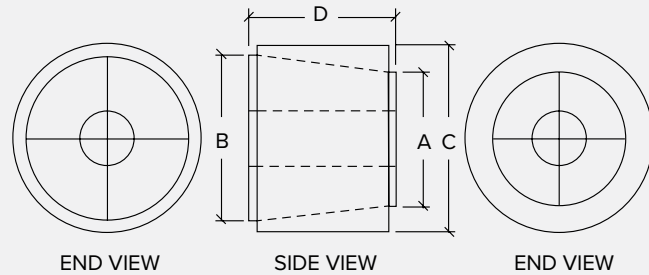
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}}\right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
29	36	42	0	29	210	0.04	0.06	0.09	0.19
			0	58	345	0.04	0.06	0.08	0.18
29	36	42	12	29	280	0.04	0.07	0.06	0.19
			12	58	430	0.04	0.07	0.05	0.18
29	36	42	16	29	300	0.06	0.09	0.04	0.19
			16	58	455	0.07	0.11	0.03	0.18
29	36	42	20	29	325	0.04	0.07	0.04	0.23
			20	58	490	0.06	0.09	0.01	0.20

32-AC-D

Axial Cone
Dissipative Silencer



Insertion Loss (IL)

+ : **“forward flow”** where noise & airflow move in same direction (e.g. supply side)

- : **“reverse flow”** where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
32	0	32	- 2000	1	5	10	11	8	7	6	4
			0	1	5	10	10	8	7	6	4
			+ 2000	1	5	10	10	8	7	6	4
32	0	64	- 2000	1	7	16	20	14	11	10	9
			0	1	7	16	19	14	11	10	9
			+ 2000	1	7	16	20	14	12	11	9
32	12	32	- 2000	1	6	11	16	13	11	10	8
			0	1	5	10	15	13	11	9	8
			+ 2000	1	5	10	14	13	11	9	8
32	12	64	- 2000	2	9	18	26	23	17	14	13
			0	1	9	17	25	23	16	14	12
			+ 2000	1	9	17	25	24	17	15	14
32	16	32	- 2000	1	6	12	19	19	13	13	10
			0	1	5	11	18	18	13	12	9
			+ 2000	1	5	11	17	17	13	12	9
32	16	64	- 2000	2	10	19	31	30	22	17	16
			0	2	10	18	30	29	21	17	15
			+ 2000	2	10	18	29	29	22	18	16
32	20	32	- 2000	2	8	13	23	25	20	16	12
			0	2	7	12	21	24	20	15	12
			+ 2000	2	7	12	20	24	20	14	11
32	20	64	- 2000	2	12	21	35	35	29	22	19
			0	2	11	20	34	35	28	22	18
			+ 2000	2	11	20	33	35	29	23	19

Pressure Drop (PD)

*Note: For pressure drop at other velocities
Actual Pressure Drop =

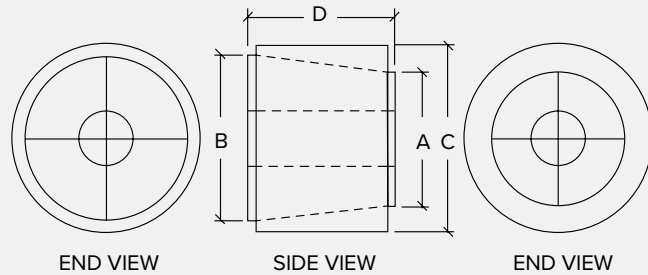
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}}\right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
32	40	46	0	32	250	0.04	0.07	0.08	0.18
			0	64	415	0.06	0.09	0.07	0.17
32	40	46	12	32	330	0.04	0.07	0.06	0.18
			12	64	510	0.06	0.09	0.06	0.17
32	40	46	16	32	350	0.03	0.05	0.04	0.18
			16	64	540	0.05	0.08	0.03	0.17
32	40	46	20	32	380	0.03	0.04	0.03	0.20
			20	64	580	0.05	0.08	0.02	0.19

36-AC-D

Axial Cone
Dissipative Silencer



Insertion Loss (IL)

+ : **“forward flow”** where noise & airflow move in same direction (e.g. supply side)

- : **“reverse flow”** where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
36	0	36	- 2000	1	4	10	9	7	7	7	4
			0	2	5	10	8	7	6	6	4
			+ 2000	1	5	10	9	7	7	6	4
36	0	72	- 2000	1	7	18	19	12	11	10	9
			0	2	7	18	18	12	11	10	9
			+ 2000	1	7	18	20	14	12	11	8
36	12	36	- 2000	1	6	11	15	12	11	10	8
			0	1	5	10	14	12	10	9	7
			+ 2000	1	5	10	14	13	11	10	7
36	12	72	- 2000	2	9	19	27	22	16	13	12
			0	1	8	18	26	22	15	13	11
			+ 2000	2	8	17	25	23	17	15	12
36	20	36	- 2000	3	7	13	21	19	15	14	13
			0	2	6	12	20	19	14	12	11
			+ 2000	1	5	12	19	19	15	13	10
36	20	72	- 2000	5	11	24	36	34	22	18	16
			0	3	10	22	35	34	21	17	14
			+ 2000	2	9	21	33	34	23	19	17
36	26	36	- 2000	4	8	15	26	26	21	16	14
			0	3	7	14	25	25	20	15	13
			+ 2000	2	8	14	24	25	21	14	12
36	26	72	- 2000	5	12	26	39	36	31	25	20
			0	4	11	24	38	36	30	25	19
			+ 2000	3	10	23	37	36	31	26	20

Pressure Drop (PD)

*Note: For pressure drop at other velocities
Actual Pressure Drop =

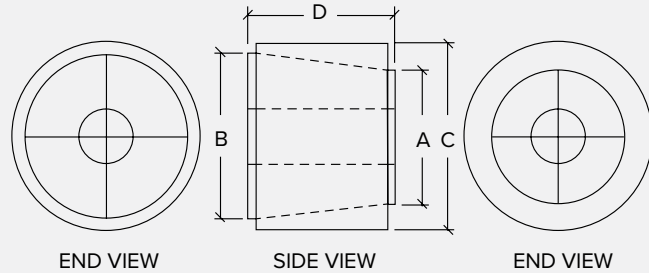
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}}\right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
36	46	51	0	36	315	0.04	0.06	0.03	0.13
			0	72	525	0.04	0.07	0.03	0.13
36	46	51	12	36	400	0.04	0.07	0.02	0.14
			12	72	630	0.05	0.08	0.02	0.14
36	46	51	20	36	455	0.03	0.04	0.06	0.21
			20	72	700	0.03	0.05	0.01	0.16
36	46	51	26	36	505	0.05	0.08	0.09	0.29
			26	72	775	0.04	0.07	0.07	0.27

42-AC-D

Axial Cone Dissipative Silencer



Insertion Loss (IL)

+ : **“forward flow”** where noise & airflow move in same direction (e.g. supply side)

- : **“reverse flow”** where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
42	0	42	-2000	2	3	9	13	9	7	6	4
			0	2	3	9	12	8	7	6	4
			+2000	2	3	9	12	9	7	6	4
42	0	84	-2000	2	6	15	23	18	11	9	9
			0	2	6	15	22	17	10	9	8
			+2000	2	6	15	22	17	11	10	8
42	16	42	-2000	3	4	11	19	19	13	11	8
			0	3	4	10	18	18	13	11	8
			+2000	3	4	9	17	17	13	11	8
42	16	84	-2000	4	7	20	26	26	19	16	12
			0	3	7	19	25	25	18	16	12
			+2000	3	7	19	25	25	18	17	13
42	20	42	-2000	4	5	13	22	22	18	14	11
			0	3	5	12	21	22	18	14	10
			+2000	3	5	11	20	21	18	15	10
42	20	84	-2000	5	9	23	29	33	25	19	16
			0	4	8	22	28	32	25	19	16
			+2000	4	8	21	28	32	25	20	17
42	26	42	-2000	4	7	14	26	30	21	20	13
			0	3	6	13	25	30	20	20	12
			+2000	3	6	12	24	29	20	21	12
42	26	84	-2000	5	11	25	34	37	33	25	20
			0	4	10	24	33	36	33	25	19
			+2000	4	10	24	33	36	33	26	20

Pressure Drop (PD)

*Note: For pressure drop at other velocities
Actual Pressure Drop =

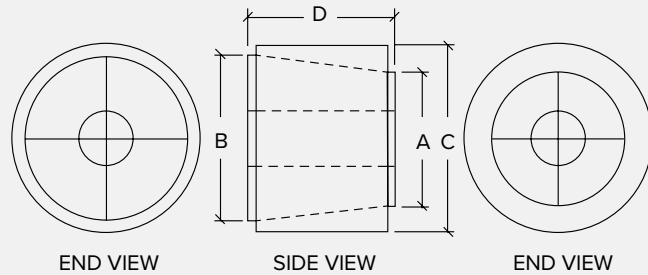
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}}\right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
42	54	59	0	42	425	0.04	0.06	0.03	0.13
			0	84	720	0.04	0.07	0.03	0.13
42	54	59	16	42	550	0.03	0.05	0.03	0.15
			16	84	875	0.04	0.06	0.03	0.15
42	54	59	20	42	585	0.03	0.05	0.05	0.19
			20	84	920	0.04	0.06	0.04	0.18
42	54	59	26	42	640	0.06	0.10	0.04	0.21
			26	84	995	0.06	0.10	0.02	0.19

48-AC-D

Axial Cone
Dissipative Silencer



Insertion Loss (IL)

+ : **“forward flow”** where noise & airflow move in same direction (e.g. supply side)

- : **“reverse flow”** where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
48	0	48	-2000	2	3	9	12	9	7	6	4
			0	2	3	9	12	8	7	6	4
			+2000	2	3	9	12	9	7	6	4
48	0	96	-2000	2	6	14	21	18	11	9	9
			0	2	6	14	20	17	10	9	8
			+2000	2	6	14	20	17	11	10	8
48	16	48	-2000	2	4	9	15	15	11	9	7
			0	2	4	9	14	14	11	9	7
			+2000	2	4	8	13	13	11	9	7
48	16	96	-2000	3	7	16	23	21	16	13	11
			0	3	6	15	22	20	15	13	11
			+2000	3	6	15	22	20	15	13	11
48	20	48	-2000	3	4	11	18	19	14	11	9
			0	3	4	10	17	18	14	11	9
			+2000	3	4	9	16	17	15	11	9
48	20	96	-2000	4	7	20	25	26	19	16	12
			0	3	7	19	24	25	18	16	12
			+2000	3	7	19	24	25	18	17	13
48	26	48	-2000	4	5	12	21	22	18	14	12
			0	3	5	11	20	22	17	14	12
			+2000	3	5	10	19	21	17	15	11
48	26	96	-2000	5	9	23	28	33	25	19	16
			0	4	8	22	27	32	24	19	16
			+2000	4	8	21	27	32	24	20	17

Pressure Drop (PD)

*Note: For pressure drop at other velocities
Actual Pressure Drop =

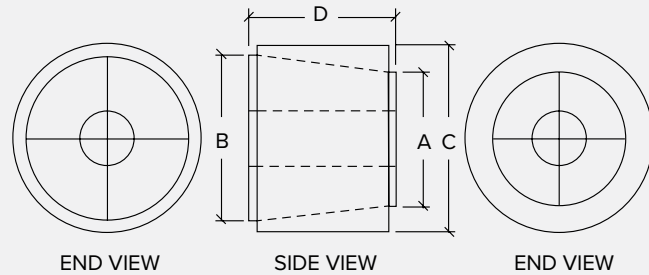
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}}\right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
48	60	66	0	48	560	0.03	0.05	0.03	0.13
			0	96	950	0.04	0.06	0.03	0.13
48	60	66	16	48	700	0.04	0.06	0.03	0.15
			16	96	1125	0.04	0.07	0.03	0.15
48	60	66	20	48	735	0.03	0.05	0.05	0.18
			20	96	1175	0.04	0.06	0.04	0.17
48	60	66	26	48	795	0.06	0.10	0.06	0.20
			26	96	1255	0.06	0.10	0.06	0.20

54-AC-D

Axial Cone
Dissipative Silencer



Insertion Loss (IL)

+ : **“forward flow”** where noise & airflow move in same direction (e.g. supply side)

- : **“reverse flow”** where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
54	0	54	-2000	2	3	9	13	9	7	6	4
			0	2	3	9	12	8	7	6	4
			+2000	2	3	9	12	9	7	6	4
54	0	108	-2000	2	6	13	21	18	11	9	9
			0	2	6	13	20	17	10	9	8
			+2000	2	6	13	20	17	11	10	8
54	16	54	-2000	2	4	9	14	15	9	9	7
			0	2	4	9	13	14	9	9	7
			+2000	2	4	8	12	13	9	9	7
54	16	108	-2000	3	7	15	22	21	15	13	11
			0	3	6	14	21	20	14	13	11
			+2000	3	6	14	21	20	14	13	11
54	20	54	-2000	3	4	10	17	19	13	11	9
			0	3	4	9	16	18	13	11	9
			+2000	3	4	9	15	17	13	11	9
54	20	108	-2000	4	7	19	24	26	19	16	12
			0	3	7	18	23	25	18	16	12
			+2000	3	7	18	23	25	18	17	13
54	26	54	-2000	4	5	12	20	22	18	14	12
			0	3	5	11	19	22	18	14	12
			+2000	3	5	10	18	21	18	15	11
54	26	108	-2000	5	9	22	27	33	25	19	16
			0	4	8	21	26	32	24	19	16
			+2000	4	8	20	26	32	24	20	17

Pressure Drop (PD)

*Note: For pressure drop at other velocities
Actual Pressure Drop =

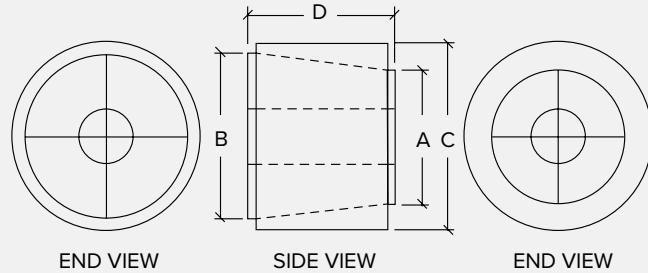
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}}\right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
54	68	74	0	54	710	0.03	0.05	0.03	0.13
			0	108	1215	0.04	0.06	0.03	0.13
54	68	74	16	54	870	0.04	0.06	0.03	0.14
			16	108	1415	0.04	0.07	0.03	0.14
54	68	74	20	54	905	0.03	0.05	0.05	0.17
			20	108	1465	0.04	0.06	0.05	0.17
54	68	74	26	54	970	0.04	0.07	0.05	0.19
			26	108	1555	0.05	0.08	0.05	0.19

60-AC-D

Axial Cone
Dissipative Silencer



Insertion Loss (IL)

+ : **“forward flow”** where noise & airflow move in same direction (e.g. supply side)

- : **“reverse flow”** where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
60	0	60	-2000	1	4	8	13	9	7	6	4
			0	1	4	8	12	8	7	6	4
			+2000	1	4	7	12	9	7	6	4
60	0	120	-2000	2	7	13	21	18	11	9	9
			0	2	7	13	20	17	10	9	8
			+2000	2	7	12	20	17	11	10	8
60	20	60	-2000	1	5	9	14	15	11	9	7
			0	1	4	8	13	14	11	9	7
			+2000	1	4	7	12	13	11	9	7
60	20	120	-2000	4	8	17	22	21	16	13	11
			0	3	8	16	21	20	15	13	11
			+2000	3	8	17	21	20	15	13	11
60	26	60	-2000	2	5	11	17	19	14	11	9
			0	2	4	10	16	18	14	11	9
			+2000	2	4	10	15	17	14	11	9
60	26	120	-2000	5	10	21	24	26	19	16	12
			0	4	9	20	23	25	18	16	12
			+2000	4	9	20	23	25	18	17	13
60	30	60	-2000	4	8	14	21	25	18	14	12
			0	3	7	13	20	25	17	14	12
			+2000	3	7	12	19	21	17	14	12
60	30	120	-2000	5	12	24	27	28	21	17	14
			0	4	11	23	26	27	20	17	14
			+2000	5	11	23	25	27	20	18	14

Pressure Drop (PD)

*Note: For pressure drop at other velocities
Actual Pressure Drop =

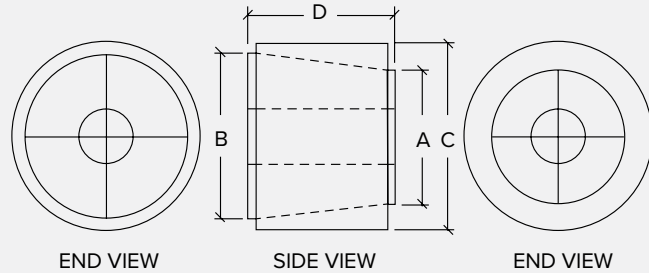
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}}\right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
60	76	81	0	60	880	0.03	0.05	0.03	0.13
			0	120	1510	0.04	0.06	0.03	0.13
60	76	81	20	60	1100	0.03	0.05	0.03	0.15
			20	120	1790	0.04	0.06	0.03	0.15
60	76	81	26	60	1160	0.04	0.06	0.05	0.18
			26	120	1885	0.04	0.07	0.04	0.17
60	76	81	30	60	1215	0.04	0.06	0.06	0.20
			30	120	1960	0.05	0.08	0.05	0.19

66-AC-D

Axial Cone
Dissipative Silencer



Insertion Loss (IL)

+ : **“forward flow”** where noise & airflow move in same direction (e.g. supply side)

- : **“reverse flow”** where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
66	0	66	-2000	1	4	8	13	9	7	6	4
			0	1	4	8	12	8	7	6	4
			+2000	1	4	7	12	9	7	6	4
66	0	99	-2000	2	7	13	21	18	11	9	9
			0	2	7	13	20	17	10	9	8
			+2000	2	7	12	20	17	11	10	8
66	20	66	-2000	1	5	9	14	15	11	9	7
			0	1	4	8	13	14	11	9	7
			+2000	1	4	7	12	13	11	9	7
66	20	99	-2000	4	8	19	22	21	16	13	11
			0	3	8	18	21	20	15	13	11
			+2000	3	8	19	21	20	15	13	11
66	26	66	-2000	2	5	10	17	19	14	11	9
			0	2	4	9	16	18	14	11	9
			+2000	2	4	9	15	17	14	11	9
66	26	99	-2000	5	10	22	24	26	19	16	12
			0	4	9	21	23	25	18	16	12
			+2000	4	9	21	23	25	18	17	13
66	30	66	-2000	4	8	14	21	25	18	14	12
			0	3	7	13	20	25	17	14	12
			+2000	3	7	12	19	24	17	14	12
66	30	99	-2000	5	12	25	27	28	21	17	14
			0	4	11	24	26	27	20	17	14
			+2000	5	11	24	25	27	20	17	14

Pressure Drop (PD)

*Note: For pressure drop at other velocities
Actual Pressure Drop =

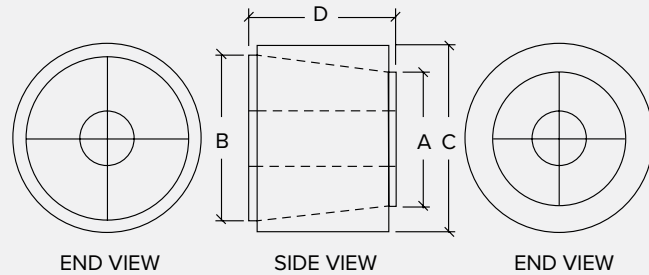
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}}\right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
66	84	89	0	66	1075	0.03	0.05	0.03	0.13
			0	99	1465	0.04	0.06	0.03	0.13
66	84	89	20	66	1315	0.03	0.05	0.03	0.15
			20	99	1740	0.04	0.06	0.03	0.15
66	84	89	26	66	1385	0.04	0.06	0.05	0.18
			26	99	1825	0.05	0.08	0.05	0.18
66	84	89	30	66	1435	0.04	0.06	0.06	0.20
			30	99	1890	0.05	0.08	0.05	0.19

72-AC-D

Axial Cone
Dissipative Silencer



Insertion Loss (IL)

+ : **“forward flow”** where noise & airflow move in same direction (e.g. supply side)

- : **“reverse flow”** where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
72	0	72	-2000	1	4	8	11	9	7	6	4
			0	1	4	8	11	8	7	6	4
			+2000	1	4	7	11	9	7	6	4
72	0	108	-2000	2	7	13	19	18	10	9	10
			0	2	7	12	18	17	10	9	9
			+2000	2	7	12	18	17	10	10	9
72	20	72	-2000	1	5	12	14	15	13	9	7
			0	1	4	11	13	14	12	9	7
			+2000	1	4	11	12	13	13	9	7
72	20	108	-2000	4	8	17	21	21	18	12	11
			0	3	8	16	20	20	17	11	11
			+2000	3	8	17	20	20	18	12	11
72	26	72	-2000	2	5	15	16	19	15	11	10
			0	2	4	14	15	18	14	11	10
			+2000	2	4	14	15	17	14	11	10
72	26	108	-2000	5	10	21	23	26	20	14	12
			0	4	9	20	22	25	19	13	12
			+2000	4	9	20	22	25	20	14	12
72	30	72	-2000	4	8	18	17	25	17	14	12
			0	3	7	17	16	25	16	14	11
			+2000	3	7	16	16	24	16	14	11
72	30	108	-2000	5	12	24	27	28	22	16	13
			0	4	11	23	26	27	21	15	12
			+2000	5	11	23	25	27	21	16	12

Pressure Drop (PD)

*Note: For pressure drop at other velocities
Actual Pressure Drop =

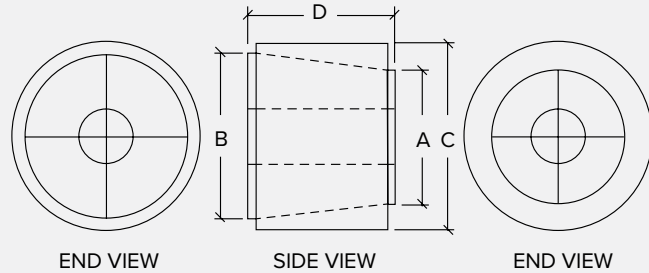
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}}\right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
72	90	96	0	72	1295	0.03	0.05	0.03	0.13
			0	108	1770	0.04	0.06	0.03	0.13
72	90	96	20	72	1560	0.03	0.05	0.04	0.15
			20	108	2070	0.04	0.06	0.04	0.15
72	90	96	26	72	1630	0.03	0.05	0.06	0.18
			26	108	2160	0.04	0.06	0.06	0.18
72	90	96	30	72	1685	0.04	0.06	0.07	0.20
			30	108	2230	0.04	0.07	0.06	0.19

78-AC-D

Axial Cone
Dissipative Silencer



Insertion Loss (IL)

+ : **“forward flow”** where noise & airflow move in same direction (e.g. supply side)

- : **“reverse flow”** where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
78	0	78	-2000	2	5	8	13	8	7	6	4
			0	2	5	8	12	8	7	6	4
			+2000	2	5	7	12	8	7	6	4
78	0	117	-2000	3	8	10	20	17	12	9	8
			0	3	8	10	19	16	12	9	8
			+2000	3	8	9	19	16	12	10	8
78	20	78	-2000	2	6	10	14	14	12	9	7
			0	2	5	9	13	13	12	9	7
			+2000	2	5	8	12	12	12	9	7
78	20	117	-2000	5	9	14	21	19	16	13	10
			0	5	9	13	20	18	15	13	10
			+2000	4	9	13	20	18	15	13	10
78	26	78	-2000	3	6	11	16	17	14	10	9
			0	3	5	10	15	16	14	10	9
			+2000	3	5	10	14	15	14	10	9
78	26	117	-2000	6	11	16	23	24	19	16	11
			0	5	10	15	22	23	18	16	11
			+2000	5	10	15	22	23	18	17	11
78	30	78	-2000	5	9	13	20	23	16	13	10
			0	4	8	12	19	23	16	13	10
			+2000	4	8	11	18	22	16	13	10
78	30	117	-2000	6	13	18	26	26	21	17	11
			0	6	12	17	25	25	20	17	11
			+2000	6	12	17	24	25	20	17	11

Pressure Drop (PD)

*Note: For pressure drop at other velocities
Actual Pressure Drop =

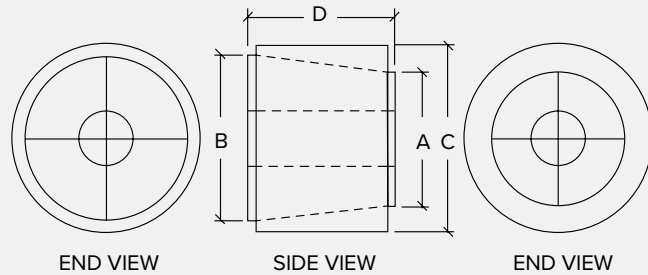
$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}}\right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
78	98	104	0	78	1560	0.03	0.05	0.03	0.13
			0	117	2125	0.04	0.06	0.03	0.13
78	98	104	20	78	1845	0.03	0.05	0.04	0.15
			20	117	1255	0.04	0.06	0.04	0.15
78	98	104	26	78	1920	0.04	0.06	0.06	0.18
			26	117	2545	0.04	0.07	0.06	0.18
78	98	104	30	78	1975	0.04	0.06	0.06	0.18
			30	117	2620	0.04	0.07	0.06	0.18

84-AC-D

Axial Cone
Dissipative Silencer



Insertion Loss (IL)

+ : "forward flow" where noise & airflow move in same direction (e.g. supply side)

- : "reverse flow" where noise & airflow move in opposite directions (e.g. return side)

CD: Centerbody Diameter

*Note: The centerbody diameter should be matched to the fan hub diameter for an inlet silencer or the fan motor diameter for a discharge silencer.

See [Silencer Selection Instructions](#). DIL above 50dB may be limited due to noise flanking around the silencer or along the duct walls. If more than 50dB DIL is required, contact your local Vibro-Acoustics representative or call **1-800-565-8401**.

A (in.)	CD*	D (in.)	Fan Inlet/Outlet Velocity (ft. per min.)	Octave Band - Hz/Dynamic Insertion Loss (dB)							
				63	125	250	500	1000	2000	4000	8000
84	0	84	-2000	2	5	8	13	8	6	6	4
			0	2	5	8	12	8	6	6	4
			+2000	2	5	7	12	8	6	6	4
84	0	126	-2000	3	8	11	20	16	10	9	9
			0	3	8	10	19	15	10	9	8
			+2000	3	8	9	19	15	10	10	8
84	20	84	-2000	2	6	9	14	13	11	9	6
			0	2	5	8	13	12	11	9	6
			+2000	2	5	7	12	11	11	9	6
84	20	126	-2000	5	9	11	21	20	15	13	9
			0	5	9	10	20	19	14	13	9
			+2000	4	9	10	20	19	14	13	9
84	26	84	-2000	3	6	9	16	16	13	10	8
			0	3	5	9	16	15	13	10	8
			+2000	3	5	9	15	14	13	10	8
84	26	126	-2000	6	11	14	22	23	18	16	11
			0	5	10	13	21	22	17	16	10
			+2000	5	10	13	21	22	17	17	10
84	30	84	-2000	5	9	12	20	22	15	13	10
			0	4	8	11	19	22	15	13	9
			+2000	4	8	10	18	21	15	13	9
84	30	126	-2000	6	13	16	25	25	19	17	11
			0	6	12	15	24	24	19	17	10
			+2000	6	12	15	23	24	19	17	10

Pressure Drop (PD)

*Note: For pressure drop at other velocities
Actual Pressure Drop =

$$\left(\frac{\text{Actual Velocity}}{2000 \text{ FPM}}\right)^2 \times \text{PD from chart}$$

Pressure drops are reported in accordance with ASTM E477 methods and are based upon **ideal** flow conditions (5 diameters of straight duct on silencer inlet and 10 on outlet). Less than ideal conditions will result in an increase in pressure drop due to System Effects. See [Silencer System Effects Data](#).

A (in.)	B (in.)	C (in.)	Centerbody Diameter (in.)	D (in.)	Weight	Pressure Drop based on Fan Inlet/Outlet Velocity = 2000 FPM*			
						Silencer On:			
						Fan Inlet		Fan Outlet	
						Ducted	Unducted	Ducted	Unducted
84	106	111	0	84	1820	0.03	0.05	0.03	0.13
			0	126	2490	0.04	0.06	0.03	0.13
84	106	111	20	84	2130	0.03	0.05	0.04	0.15
			20	126	2845	0.04	0.06	0.04	0.15
84	106	111	26	84	2210	0.04	0.06	0.05	0.17
			26	126	2945	0.04	0.07	0.05	0.17
84	106	111	30	84	2270	0.04	0.06	0.06	0.18
			30	126	3020	0.04	0.07	0.06	0.18